

Fitness For Service Assessment

API 579 ASME FFS 1

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What is Fitness for Service Assessment

- FFS is a post construction assessment for in-service equipment
- Produces an output with quantitative engineering analysis to demonstrate structural integrity
- FFS aids engineering judgment for equipment having --
 - ▶ Presence of crack like flaw/ thinning/ distortion/ creep damage
 - ▶ Change in material properties / metallurgical degradation
 - ▶ Concerns on not meeting current design standards or best practices
 - ▶ Concerns on fault scenarios (unavoidable process upsets)
 - ▶ Changes in operating parameters which are more onerous than current
 - ▶ Operation under high temperature creep environment
 - ▶ Operation under mechanical or thermal fatigue environment

Need for Fitness for Service

- ASME, API, BS 5500 & other recognized Design codes provide rules for design and fabrication of new fabrications
 - ▶ Design consideration mainly covers corrosion allowance or creep rupture life
- Acceptable flaws during construction is based on “Quality Control Levels”.
 - ▶ Quality Control levels are usually both arbitrary and conservative, but are of considerable value as they provide a route to achieve reasonable consistency and confidence in the quality of the fabricated item.
 - ▶ Any of previously acceptable flaw can grow during service and it fall beyond acceptable level as per new construction quality standards
- Post construction, none of the fabrication codes addresses deterioration during operation, such as presence of a crack, metal loss at localized area, creep damage or mechanical damage like distortion or dents.
 - ▶ FFS assessment helps the management to take decision on action against such flaws.

Which equipment are assessed for FFS

- API579 ASME FFS-1 provides assessment guideline for static equipment
 - ▶ Pressurized and non-pressurized vessels
 - ▶ Reactors, distillation columns, absorbers, strippers, reformers, fired heaters, heat exchangers,
 - ▶ Piping and Storage tanks,
 - ▶ Utility plant items: e.g. furnace tubes, boiler drum, de-aerators, headers, economizers

FFS is applied for

- Assets lacks original designed information or it may have exceeded its useful life.



- Equipment has potential to brittle failure, low temperature



- Decommissioned equipment that may be used in different services.



- Equipment operating in creep range and cyclic services



- Operating condition that might put the integrity at risk like temperature excursions or fire damage or any corrosive condition.
- Inspection results indicate any abnormal condition that is localized metal loss, laminations, excessive pitting or damage



Codes and Standards

- FFS assessment procedures are applicable to equipment constructed to the following or equivalent codes
 - ▶ ASME B&PV Code, Section VIII, Division 1
 - ▶ ASME B&PV Code, Section VIII, Division 2
 - ▶ ASME B&PV Code, Section I
 - ▶ ASME B31.1 Piping Code
 - ▶ ASME B31.3 Piping Code
 - ▶ API 650
 - ▶ API 620

FFS Evaluation

- The FFS assessment cover present integrity of the component given a current state of damage and the projected remaining life.
- Assessment techniques provide evaluation of following flaws
 - ▶ Crack like flaw
 - ▶ Thickness loss – localized or general or pitting corrosion damage
 - ▶ Hydrogen damage
 - ▶ Creep damage
 - ▶ Fire damage – can be process overrun
 - ▶ Dents or gouges
 - ▶ Laminations in plates
 - ▶ Brittle failure

Output of Fitness for Service Assessment

- The output includes one or more of the following
 - ▶ Tolerable defect sizes and defect growth rates
 - ▶ Remaining life
 - ▶ Revised operating limits and/or other risk mitigating measures
 - ▶ Design improvements
 - ▶ Suitable NDT inspection methods and acceptable / optimized inspection interval
- Management can take important and timely decisions regarding:
 - ▶ To run item as is and at what inspection interval
 - ▶ To monitor defect and at what monitoring frequency
 - ▶ To repair or replace item and when should be carried out
 - ▶ To revise operating conditions
 - ▶ To modify design

FFS: Tool for Run-Repair decision

- When material deterioration exceeding the Quality Control levels are revealed or when material property changes / metallurgical degradation are suspected, rejection of the item may not be feasible and not necessarily automatic.
- The decisions on whether “run as is/ monitor, repair or replace” is based on the derivation of acceptance levels for defects larger than the “Quality Control levels” and / or the demonstration of suitability of materials under specific operating conditions.
 - ▶ This is the concept of Fitness-For-Service or FFS applications.
 - ▶ An item is considered to be fit for the intended service, provided it can be demonstrated (with acceptable safety margin) that the conditions to cause failure are not reached within a predetermined time period, giving due regard to the HSE and Business consequence of failure.

FFS: The approach

Before conducting the FFS assessment, it is essential to detail out:

- Investigation of the flaw – why it induced ?
 - Identification of the applicable damage mechanisms
 - Systematic and planned inspection activity of the equipment
 - Sizing of the flaw and follow-up NDE
 - Multi- disciplinary expertise to understand all related aspects
 - Calculations as per API579 ASME FFS-1 guidelines
- Decision:- Fit for service
Unfit for service, needs process alteration or repair

Role of root cause analysis in FFS

- Any of the flaws detected needs to be analyzed for the root cause:

What would be the main reason that induced the flaw:

- ▶ Improper / inferior use of material ?
 - ▶ Material degradation ?
 - ▶ Process environment ?
 - ▶ Process induced stresses / vibration / thermal cycling ?
 - ▶ Any combination of above ?
- There could be more reasons than listed above: but needs to be identified so as to eliminate its reoccurrence and reliability for future operation.

Multi-Angle Investigative Approach

- Depending on the complexity of an item & the problems, one or more expertise (multi-discipline) need to be utilized
 - ▶ Metallurgical Investigations and Root Cause Analysis
 - ▶ Stress analysis (can range from basic code calculations to Finite Element Analysis)
 - ▶ Fracture Mechanics assessments
 - ▶ Remaining life calculations
 - ▶ Assessment of acceptable and optimized Inspection Interval & Inspection Methods based on risk & consequence of failure

Role of damage mechanism identification

- After the metallurgical failure investigation, the most probable reason causing flaw becomes available however, there could exist other reasons which can also induce such flaw
- A list of probable other reasons that can induce the flaw is to be prepared.
 - ▶ E.g. flaw is a crack – what is its microscopic nature - is it filled with any corrosion product / scale? Is it branched ? Is it inter or transgranular ? What the crack tip looks like? : for every answer to this there are published different damage mechanisms.
- Identification of correct damage mechanisms that may induce a flaw provides forward path on – where to look for flaw? Where to inspect?
- Leads to Knowledge Based Inspection.

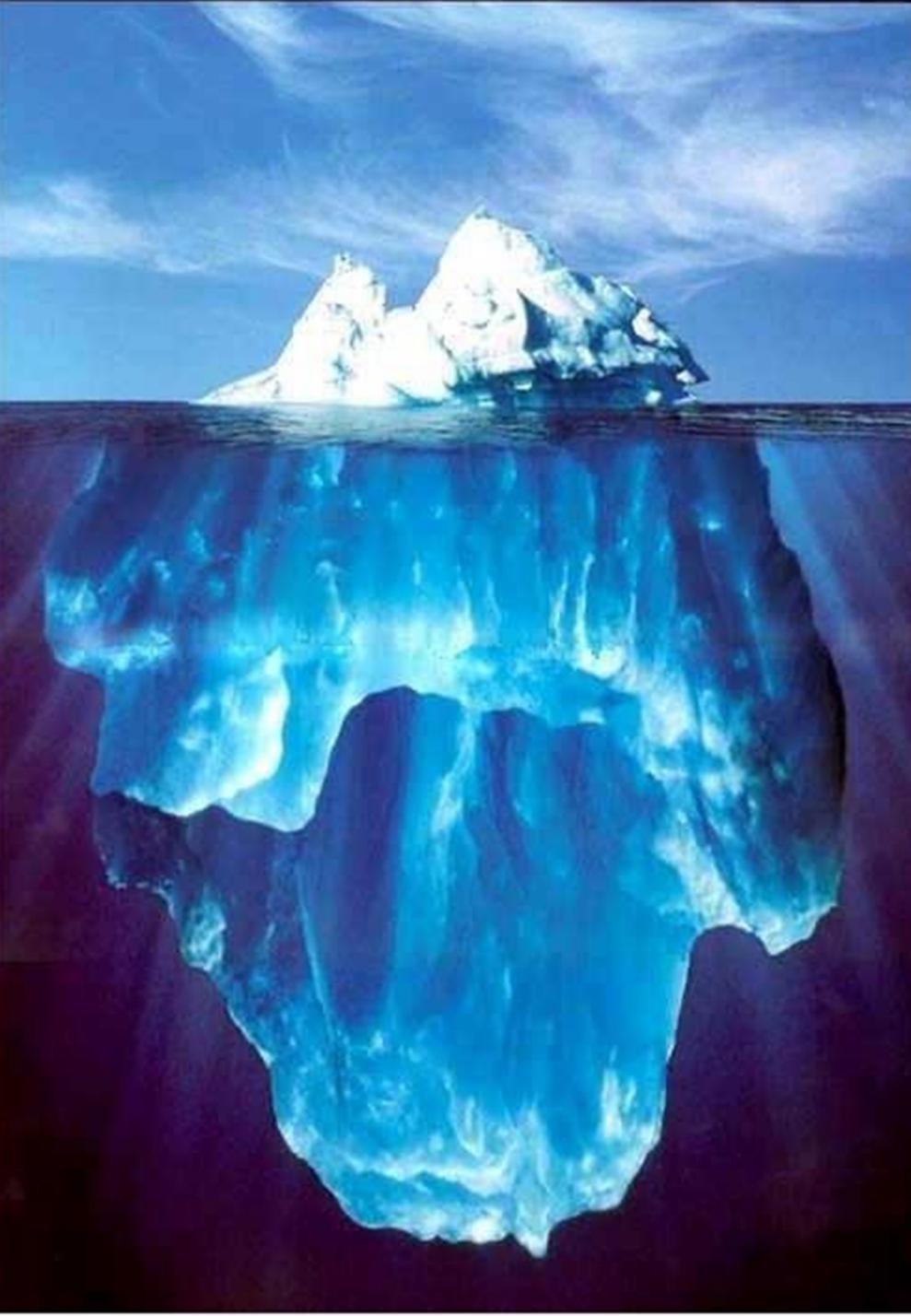
Overview of API 579 ASME FFS1

- Applicable to pressurized components in pressure vessels, piping, and tankage (principles can also be applied to rotating equipment)
- Highly structured document with a modular system based on flaw type/damage condition to facilitate use and updates
- Multi-level assessment - higher levels are less conservative but require more detailed analysis/data
 - ▶ Level 1 - Inspector/Plant Engineer
 - ▶ Level 2 - Plant Engineer
 - ▶ Level 3 - Expert Engineer

Overview of API 579

General

- General FFS assessment procedure used in API 579 for all flaw types is provided in Section 2 that includes the following steps:
 - ▶ Step 1 - Flaw & damage mechanism identification
 - ▶ Step 2 - Applicability & limitations of FFS procedures
 - ▶ Step 3 - Data requirements
 - ▶ Step 4 - Assessment techniques & acceptance criteria
 - ▶ Step 5 - Remaining life evaluation
 - ▶ Step 6 - Remediation
 - ▶ Step 7 - In-service monitoring
 - ▶ Step 8 - Documentation
- Some of the steps shown above may not be necessary depending on the application and damage mechanism

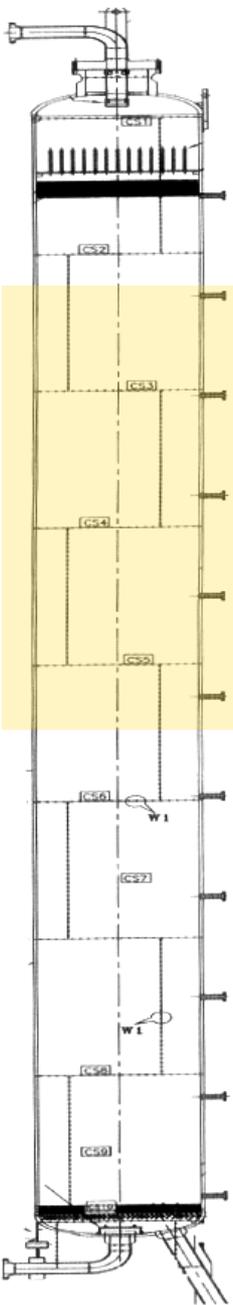


Case Studies: FFS Assessment

*Examples of Fitness-For-Service
assessment work successfully
carried out by TCR*

Operating and design parameters

Normal operating service fluid	C5 / C6 CUT + Hydrogen + Dry Hydro chloric acid	
Operating temperature	165 °C	(End of run) operating parameters (reactor outlet temperature and reactor inlet pressure)
Operating pressure	35 kg/cm ²	
Sulphur stripping operation	Hydrogen + Hydrogen sulphide + Dry Hydro chloric acid	
Operating temperature	310 °C	
Operating pressure	23.7 kg/cm ²	
Shell plate thickness	36.0 mm	
TL- TL Height	20100 mm	
Inside diameter	1600 mm	



Isomerization reactor

Location of temperature excursion

- First 4 shells from Top
- Highest temperature recorded at shell 2
- Maximum temperature recorded 710°C

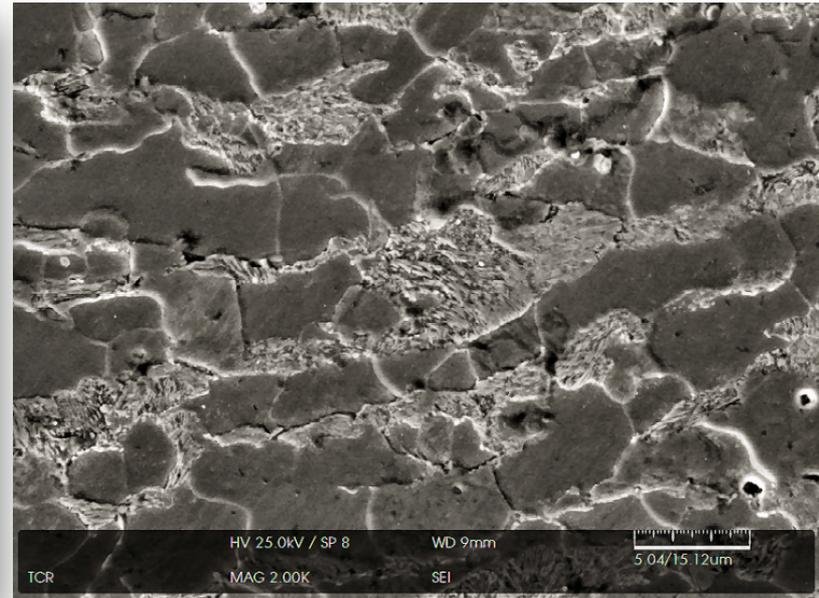
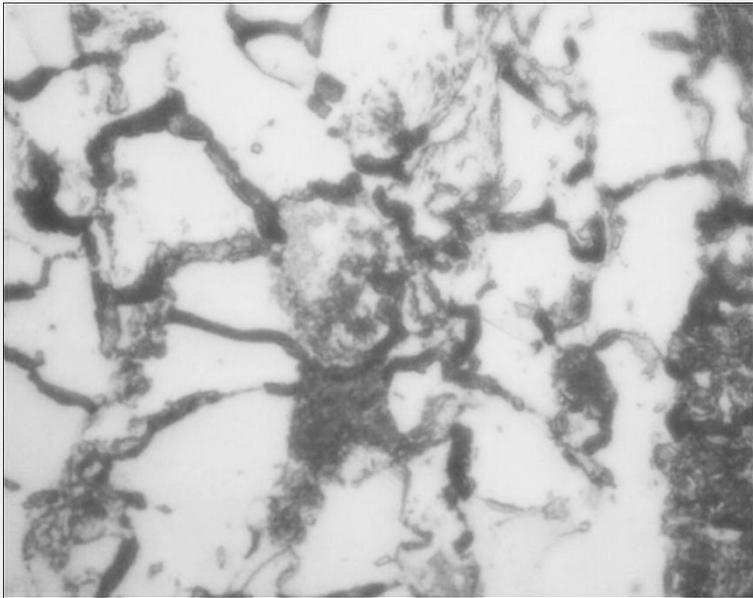
Thermocouple	Thermocouple Location	Temperature (°C)	Duration	
TW2	2 ND bed from top	710	1 min	
		>700	9 min	
		>600	44 min	
		(Design limit)	>340	3h 10min
TW3	3 RD bed from top	616	1 min	
		>600	9 min	
		(Design limit)	>340	4h 24min
		(Design limit)	>340	6h 55min
TW4	4 TH bed from top	465	1 min	
		>400	5h 26min	
		(Design limit)	>340	6h 55min

Damage mechanisms

- No operation induced damage- as it has run for 2 months.
- Anticipated damages due to accidental temperature rise :
 - High Temperature Hydrogen Attack (HTHA)
 - Metallurgical degradation of microstructure.
 - Mechanical structural distortion
 - Degradation of mechanical strength
 - High temperature corrosion
 - Integrity of weld joints

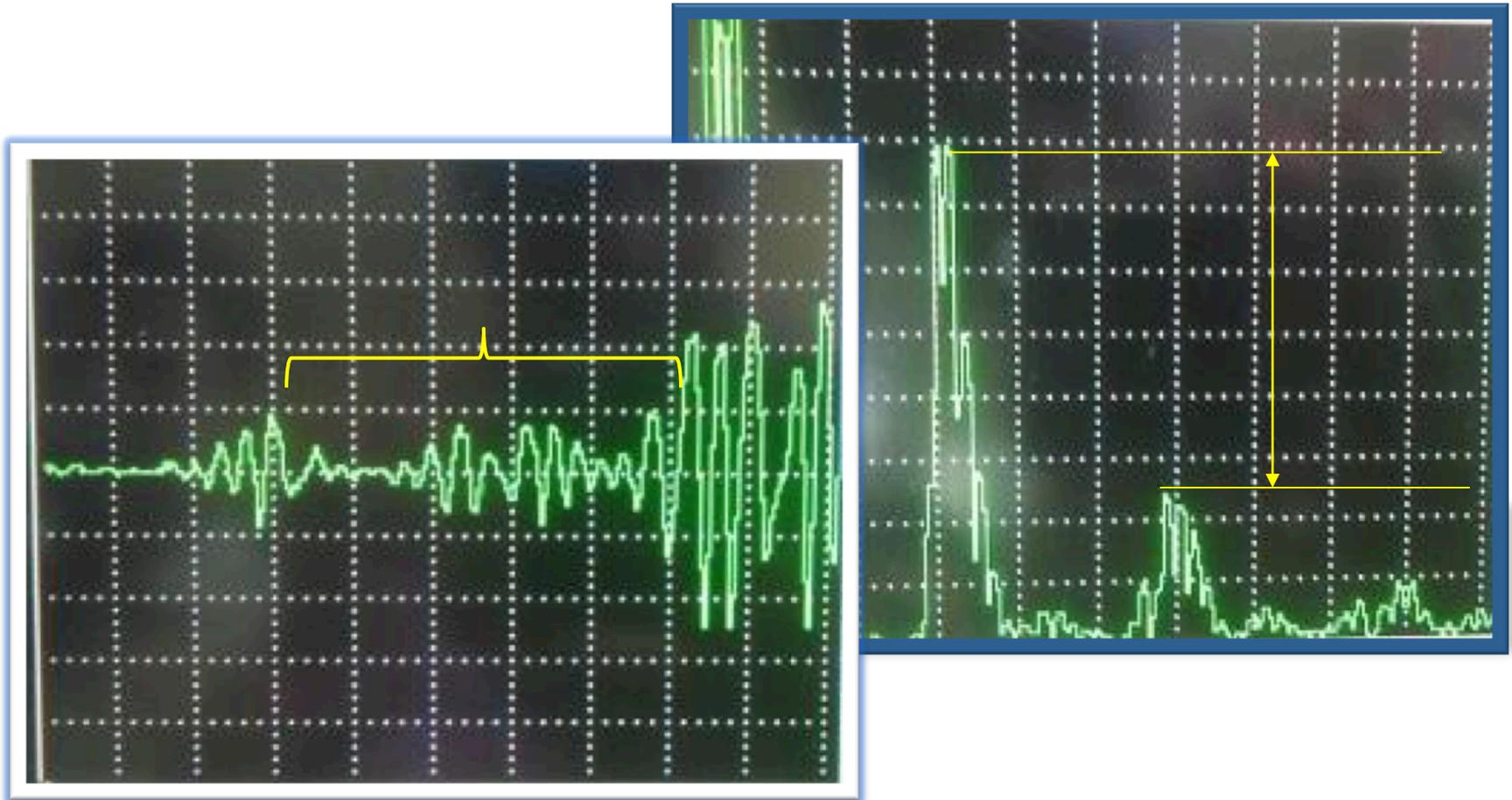
HTHA(High temperature hydrogen attack)

- Hydrogen can diffuse as nascent form in the steel
- Hydrogen reacts with cementite of pearlite in steel microstructure.
- Carbides dissociate to form methane gas (CH_4)
- Accumulated CH_4 forms micro voids and fissures at grain boundaries



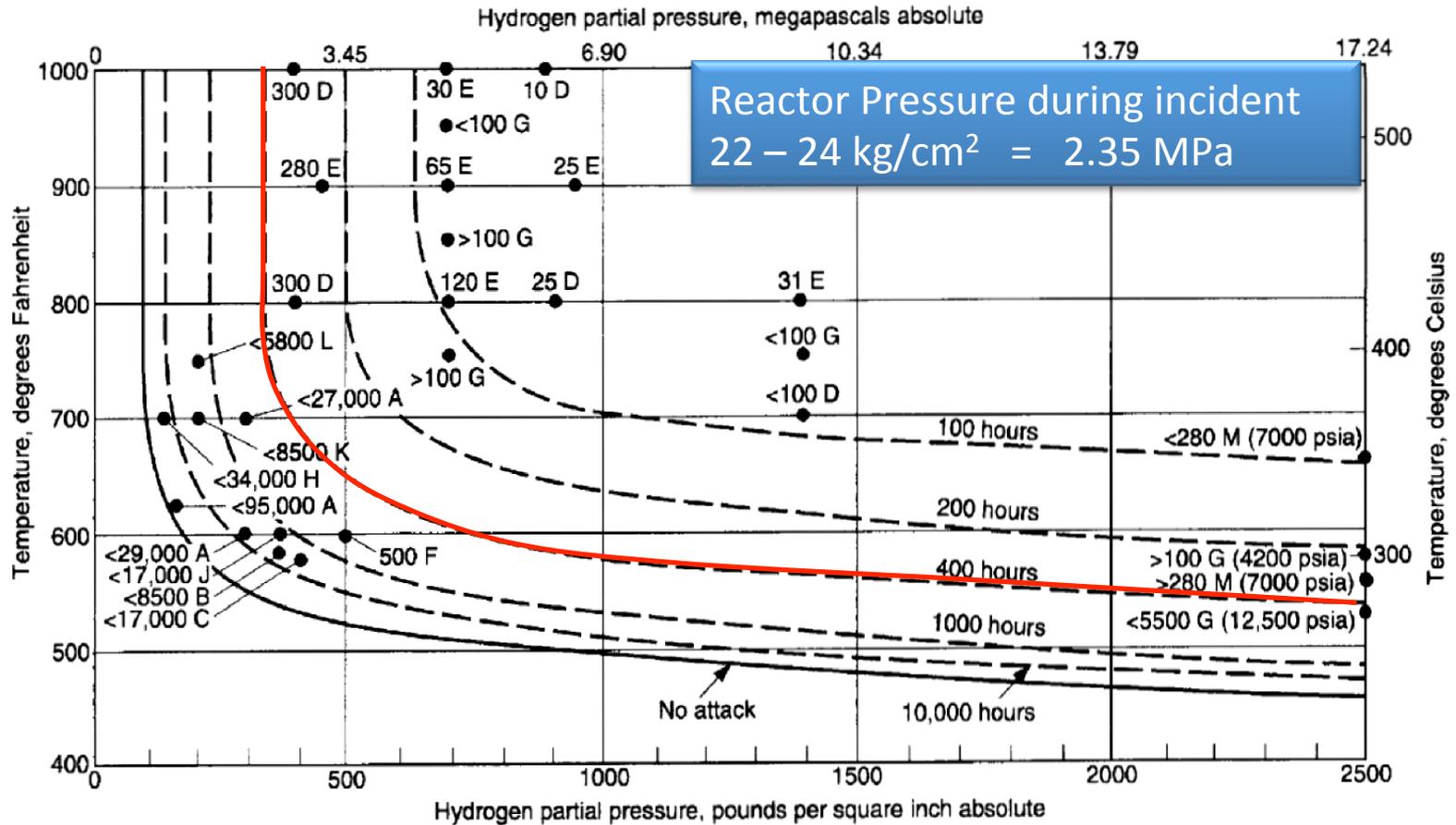
HTHA

- Detection of HTHA by Advanced Ultrasonic Backscatter Test
- Attenuation Measurements



HTHA

- Probability of HTHA based on nelson curve- API 941



Nelson's Curve : *Guideline API 941*

HTHA

Theoretical Probability of HTHA

Reactor Pressure during incident
22 – 24 kg/cm² = 341.4 PSI

The theoretical incubation period $t = C \times P^{-3} \times e^{[Q/(R \times T)]}$

Where, t: Incubation time in hours

C: constant: 1.39×10^6

P: Partial pressure of hydrogen (PSI) = 24 kg/cm² or 341.4 PSI

Q: Activation energy 14.6 kcal / mol

R: Gas constant

T: Absolute temperature of exposure (°K) = 710°C or 983°K

Gas constant for hydrogen 'R' = R_U / M_{gas}

Where, R_U : universal gas constant = 1.9858×10^{-3}

M_{Gas} : Molecular weight of H₂ (1.0079),

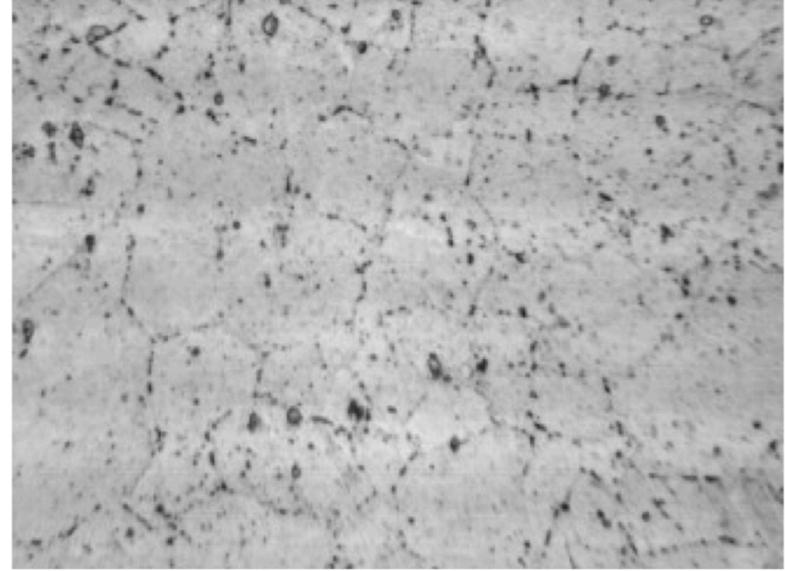
$$\begin{aligned} \text{i.e. } t &= 1.39 \times 10^6 \times 341.4^{-3} \times \text{Exp} [14.6 / (1.9702 \times 10^{-3} \times 983)] \\ &= 65.6 \text{ h} \end{aligned}$$

Metallurgical degradation

- SA516 Grade 70 in normalized conditions has of ferrite and pearlite
- Reactor shell may undergo transformation of phases if the local temperature excursion exceeds 723°C
- Pearlite gets spherodized resulting in reduction of strength



Normal structure



Spherodized pearlite

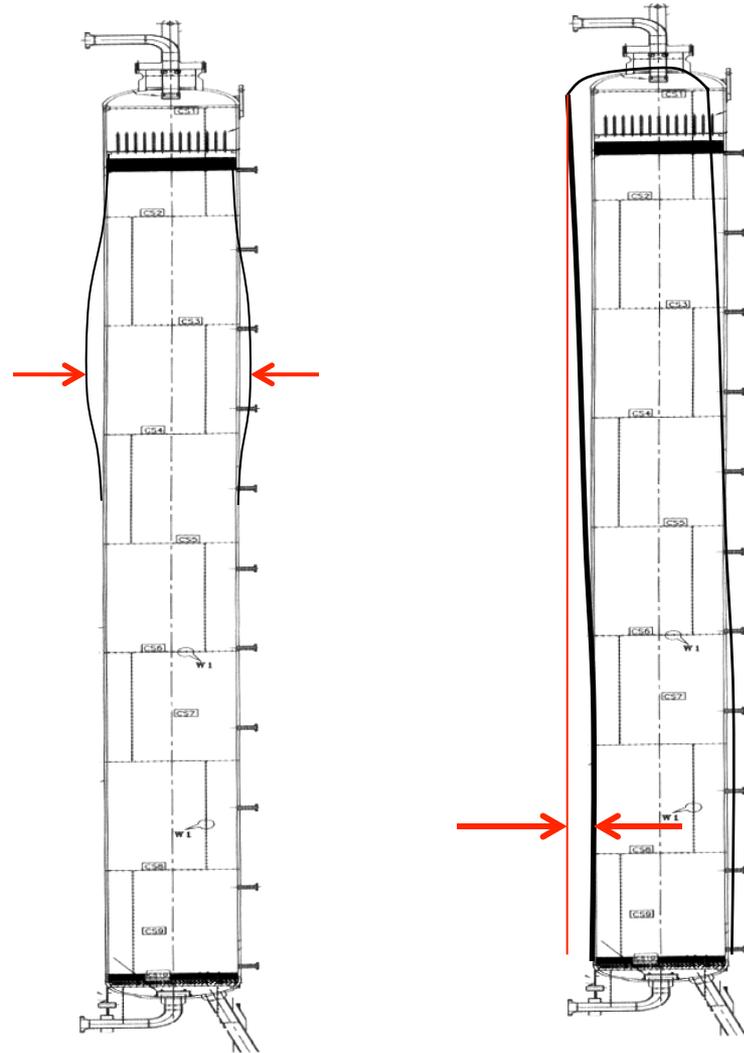
Possible structural distortion

Generally observed as

- Overall or localized bulging of reactor shell
- Leaning / out of verticality of reactor.

Dimensional verification methods:

- Change in outer diameter through circumference measurement
- Plumb measurement at 4 orientations



Other Damage Mechanisms

High temperature corrosion:

- High temperature corrosion in dry hydrochloric acid environment can cause internal damage.
- Can affect effective wall thickness and strength of material in long use
- Can be detected by ultrasonic thickness mapping.

Presence of weld flaws:

- Sudden heat excursion followed by cooling may exert high stresses at the welding joints
- At locations of high stress concentrations, internal defects like crack may occur.
- Presence of internal weld flaws can be detected through
 - Time of Flight Diffraction (TOFD) ultrasonic flaw detection
 - ‘A’ scan angle beam ultrasonic method

On-site NDT

Date of inspection	23 to 29 June 2012
Extent of coverage	All shells of reactor, all thermowell and manhole nozzles
Access for inspection	External only
Inspection techniques	Visual examination
	Outside diameter measurement
	Dimension profile of verticality
	Ultrasonic thickness measurements
	Wet Fluorescent Magnetic Particle Inspection
	TOFD Flaw Detection
	AUBT and HTHA detection
	'A' Scan – angle beaming ultrasonic flaw detection
	In-situ Metallographic Replication
	Hardness Measurements

Dimension measurement

	Outer Diameter	Tower Verticality	Shell Thickness
Total points of measurement	3 elevations on each shell	4 elevations on each shell (N, E, S, W)	2 elevations on each shell (N, E, S, W)
Observed minimum value	Circ: 5264 mm OD: 1676 mm (CS1)	6.4 mm (W)	36.6 mm (CS9)
Observed maximum value	Circ: 5275 mm OD: 1680 mm (CS8)	9.3 mm (N)	38.6 mm (W : CS3-CS4)
Maximum deviation	+4 mm Design: 1600	2.1 mm	+0.6 mm Design: 36.0 mm

No structural distortion

No effect of high temperature corrosion

Wfmpi and UT

Wet Fluorescent Magnetic Particle Inspection:

- All weld joints were subjected to 100% inspection, including the nozzles of thermowell and other insulation support joint joints
- Result: No significant linear indication observed anywhere**

‘A’ Scan Ultrasonic Flaw Detection:

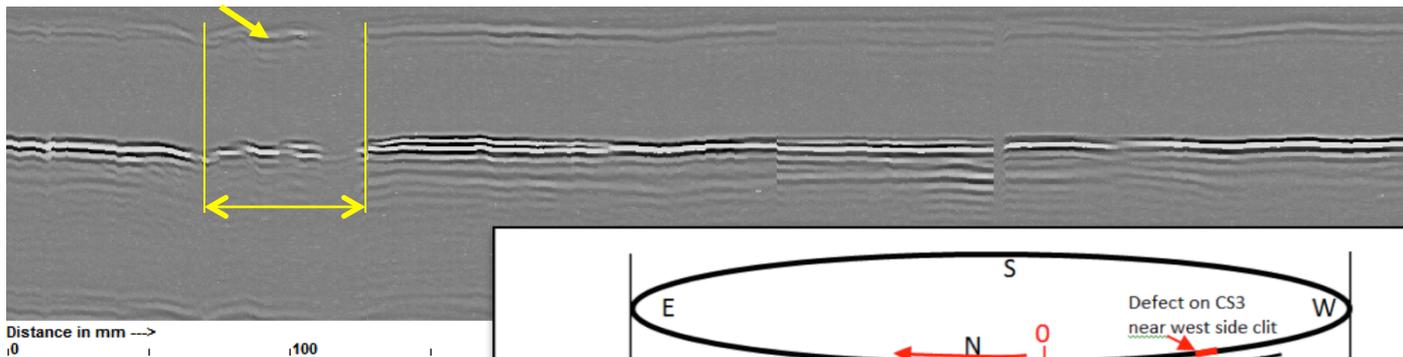
- Extent of coverage: Weld joint of CS1 and weld joints of top nozzle ‘N1’
- Probe angles : 45°, 60°
- Probe frequency: 4 MHz
- Reference :
 - V2 Block,
 - Distance Amplitude Correction on Ø4mm SDH of similar material
- Result : No significant defect indication was observed**

ToFD

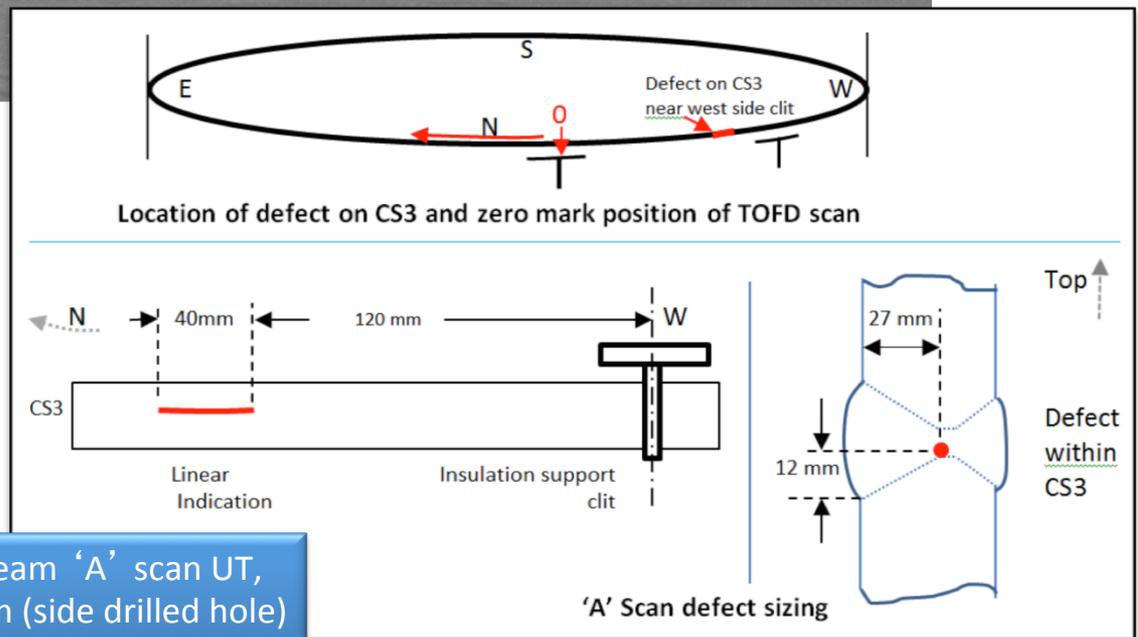
Time of Flight Diffraction (TOFD) Flaw Detection:

Extent of coverage: CS2 – CS5, LS1 – LS3, All Tee Joints

Probes: 2 MHz, Wedge Angle: 60°, Reference: ASME calibration blocks Fig 11.1 - 11.3



Drop in back wall echo with indication of flaw

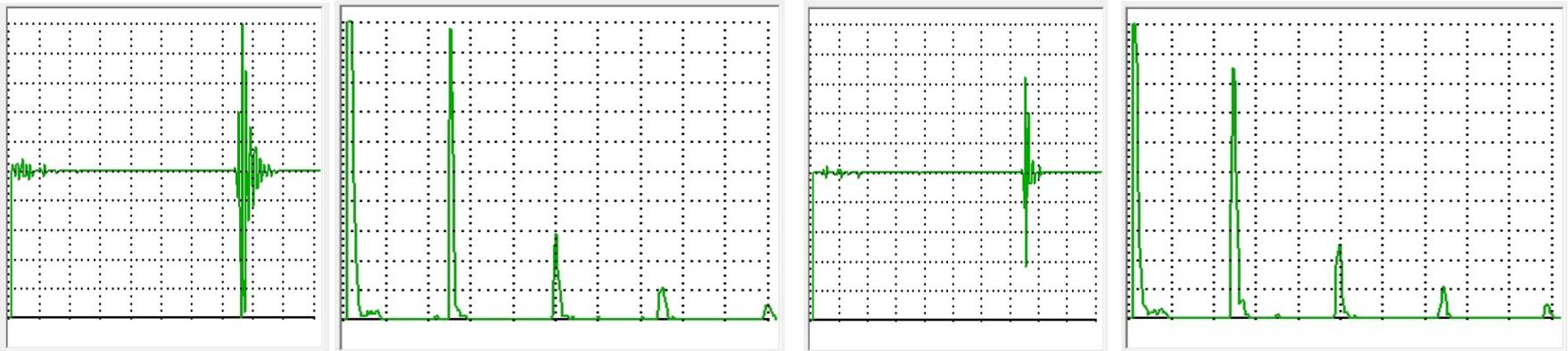


Defect sizing by angle beam 'A' scan UT, size equivalent to $\varnothing 4\text{mm}$ (side drilled hole) and 40 mm length

AUBT as per API 941

AUBT : HTHA assessment:

- Extent of coverage: First four shells: 100% scanned with 10% probe overlapping method
- Probes: 10 MHz
- References: (1) Guideline from API 941 (2) Comparison with away region
- No indication of HTHA observed anywhere



AUBT
Echo pattern at Shell 2

Attenuation

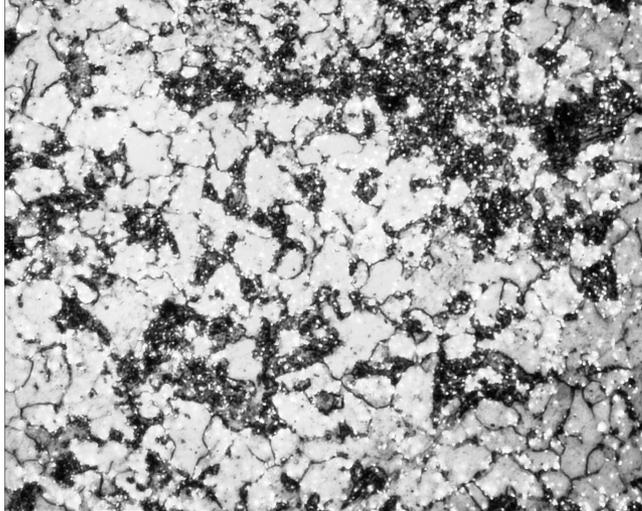
AUBT
Echo pattern Shell 8

Attenuation

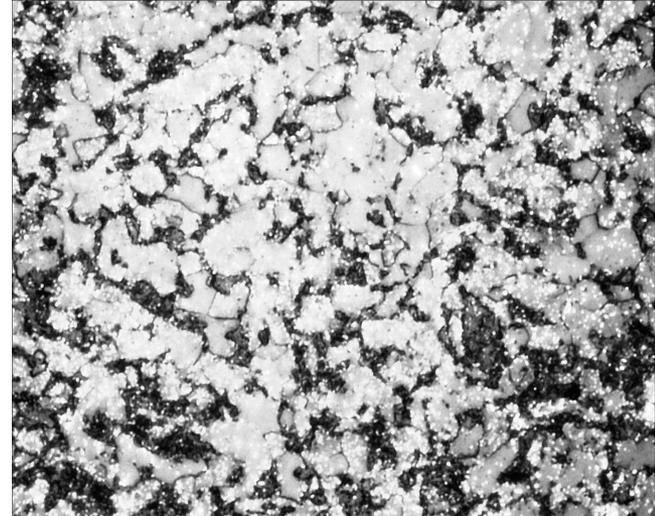
In situ metallography

In-situ metallographic replication:

- Extent of coverage: Total 60 Locations (*Shell 2 : 16 locations*)
- Method: ASTM E1351 “Practice for production and evaluation of field metallographic replicas”
- Etching technique: Manual swabbing with 2% nital
- No significant change in microstructure is observed, microstructures show ferrite and pearlite structure. ASTM Grain size 9 to 10. No indication of pearlite degradation.
- **Heat excursion on external surface of shell is insignificant**



Structure at Shell 2



Structure at Shell 8

Hardness

Hardness Measurements:

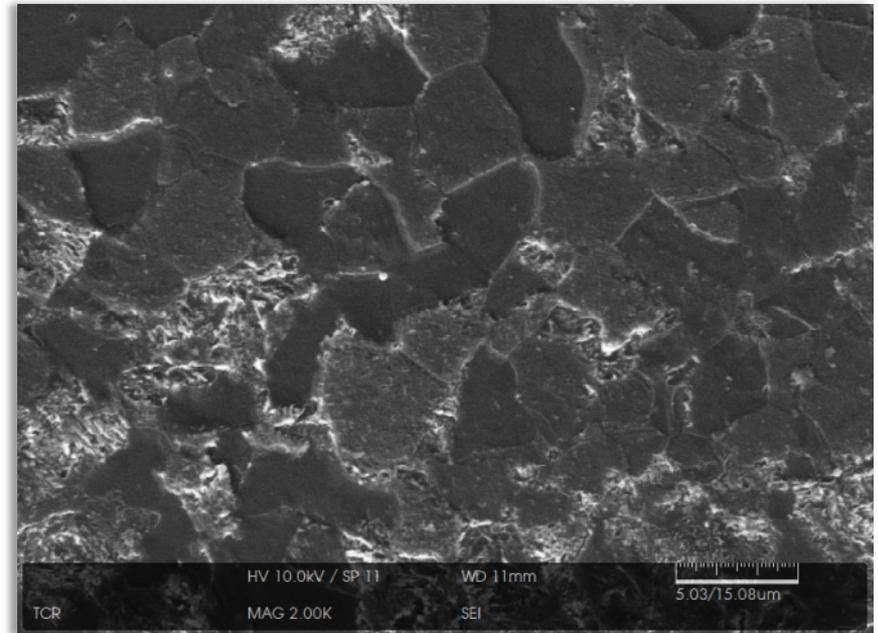
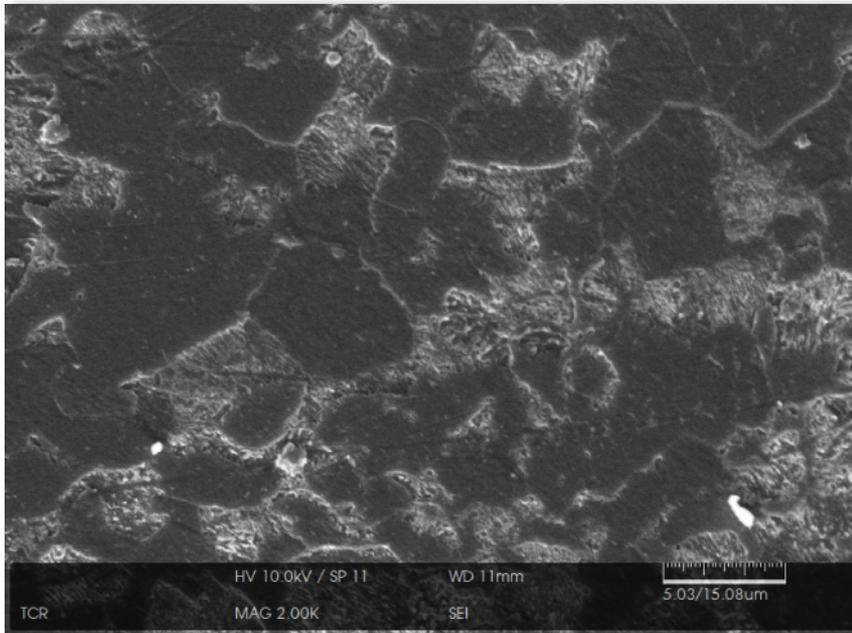
- Extent of coverage: 60 locations of metallographic replication
- Instrument used:, MIC20-Krautkramer
- Minimum Hardness: Required 147 BHN Measured : 147 BHN

Location	Minimum (BHN)	Maximum (BHN)
Overall Shell hardness range	147	188
Shell 1	148	177
Shell 2	147	170
Shell 3	150	186
Shell 4	156	188
Shell 5	155	172
Shell 6	148	168
Shell 7	151	181
Shell 8	151	169
Overall weld hardness range	162	218

Laboratory finding

Scanning Electron Microscopy (SEM) Observations:

- Extent of coverage: 15% of replicated structures
- Magnification up to 3500X after Gold coating of replica
- Finding: Fine grained ferrite and pearlite structures
No significant difference in structures

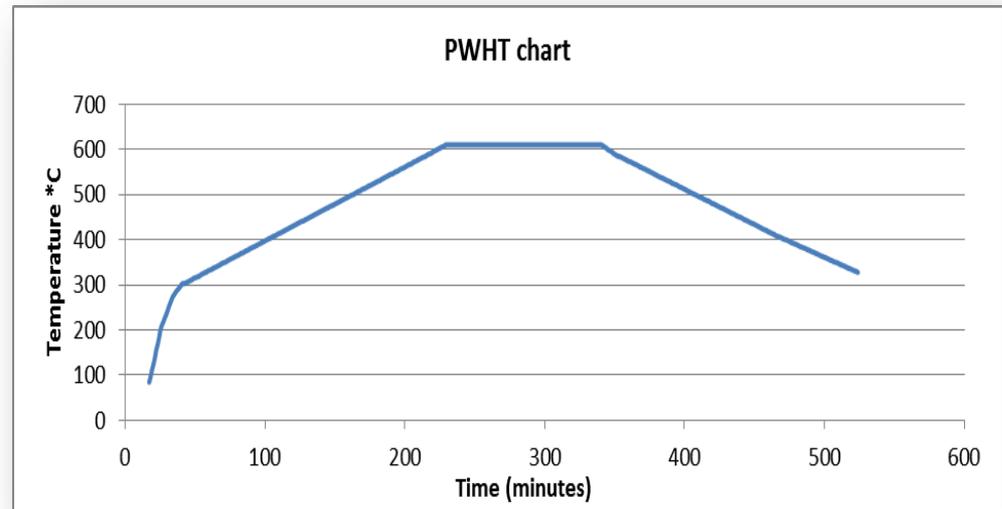
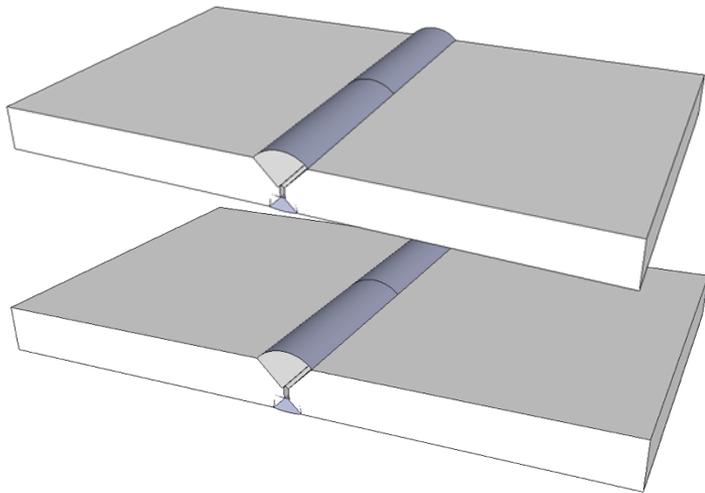


Structure from Shell 2

Structure from Shell 7

Laboratory simulation experiment

- Two 36mm thick coupon plates were prepared as per WPS given for the equipment
- Two sets of such welded pieces were fabricated at laboratory.
- Both the coupons were Post weld heat treated soaking for 2h at 610°C.

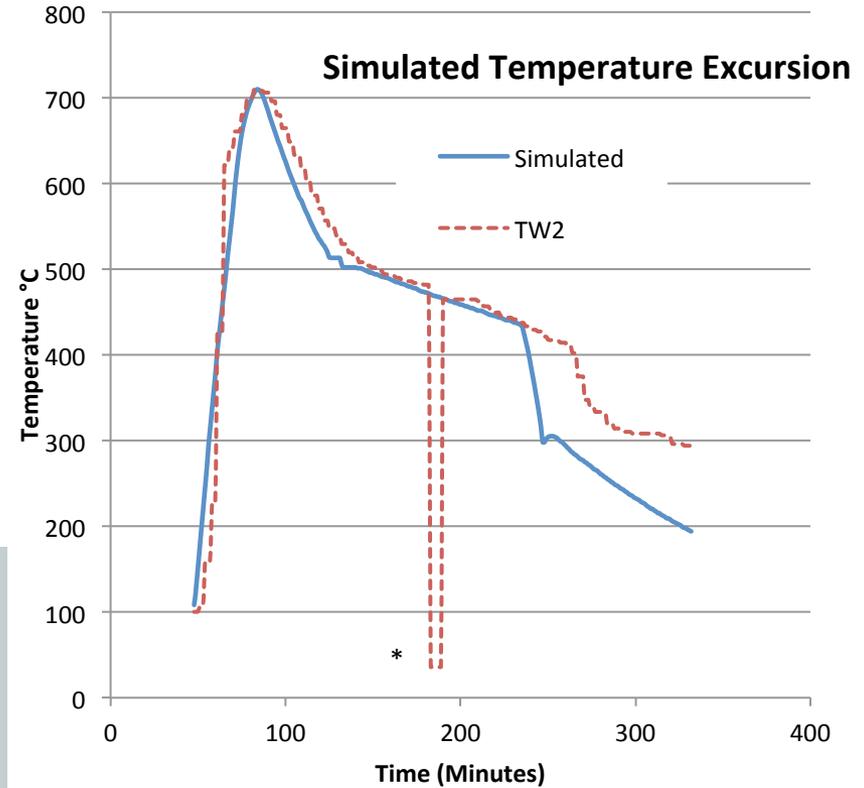
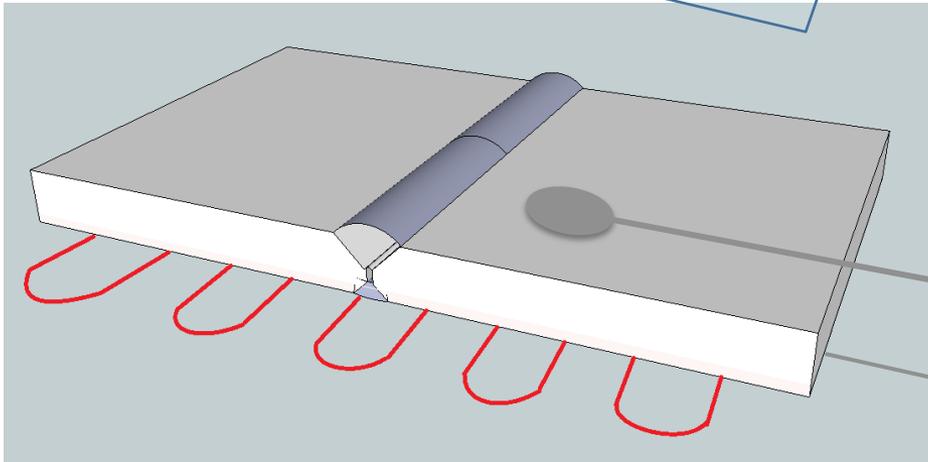


Heat excursion simulation

Welded coupon placed on heater coil

Covered with 45mm thick hot insulation

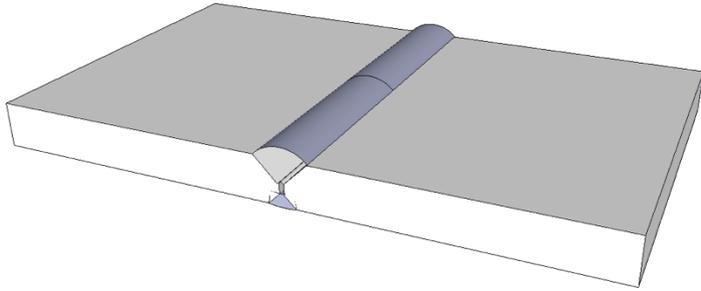
Control cooling to simulate actual heat excursion



Top thermocouple

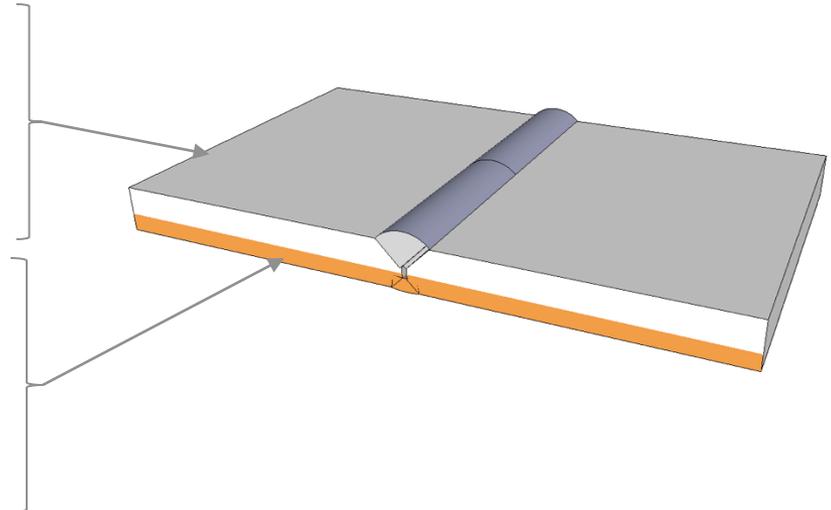
Bottom thermocouple

Mechanical tests

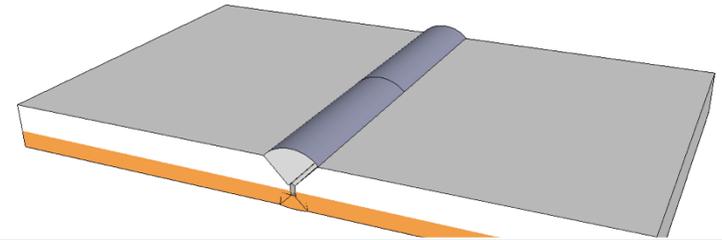


PWHT coupon test result					
	P.M.	Req.	HAZ	Weld	Req.
Y.S. (N/mm ²)	420	260	-	458	400
U.T.S. (N/mm ²)	530	485	-	535	490
E (%)	31	21	-	27.6	22
CVN (Joule)	21	20	23	146	20

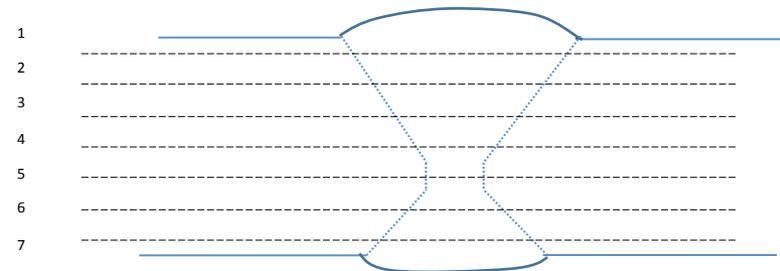
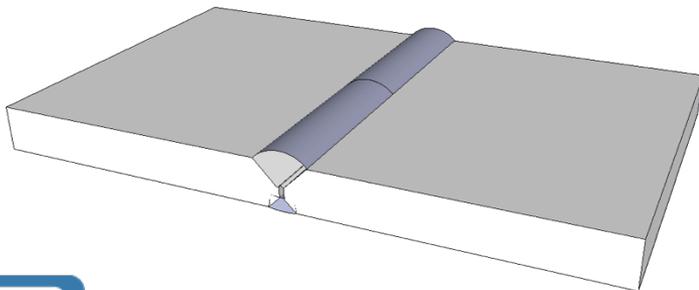
PWHT + Heat Simulated coupon test result			
	P.M.	HAZ	Weld
Y.S. (N/mm ²)	441	-	446
U.T.S. (N/mm ²)	551	-	544
E (%)	35.18	-	26.89
CVN (Joule)	67	21	113
Y.S. (N/mm²)	429	-	373
U.T.S. (N/mm ²)	558	-	474
E (%)	36.06	-	36.54
CVN (Joule)	179	28	53



Hardness (BHN)



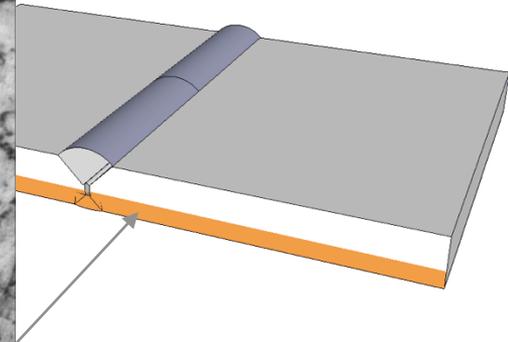
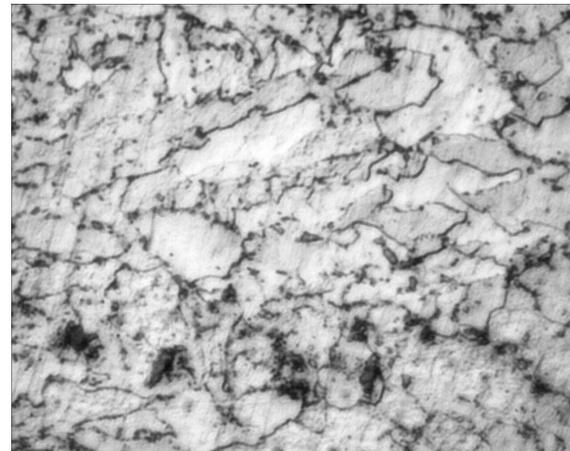
	PWHT coupon					PWHT + Heat simulated coupon				
	PM	HAZ	WELD	HAZ	PM	PM	HAZ	WELD	HAZ	PM
1	147	148	166	148	166	162	161	171	157	166
2	153	151	169	151	162	169	159	158	150	160
3	147	159	163	156	160	162	161	157	162	166
4	-	161	-	162	-	-	153	-	156	-
5	158	147	165	149	166	154	159	163	159	166
6	164	153	150	153	169	167	159	167	148	167
7	160	149	156	148	158	157	154	161	155	164
Max. Difference			10					10		



Microstructure

Between PWHT and PWHT + Heat simulated coupons

- Microstructure are of ferrite and pearlite.
- No significant change in grain size after simulated heat excursion.
- Minor effect of spheroidization of pearlite.
- No significant change in microstructural properties after short period temperature excursion up to 710°C



PWHT Coupon

PWHT + Heat simulated coupon (*Bottom*)

Fracture toughness calculation for assessment of crack like flaw in weld

However, estimated K_{IC} of SA 516 Gr 70 material (up to 200°C) as measured on notched rounded bar specimen :

$$K_{IC} \text{ (upto } 200^{\circ}\text{C)} = 136 \text{ kg/mm}$$

Q factor,

$$\text{Defect size at CS3} = (2a \times 2c) \\ = 4 \text{ mm} \times 40 \text{ mm}$$

$$a/_{2C} = 2 / 40 \\ = 0.05$$

$$\text{And, } \sigma_{\text{total}} / \sigma_{YS} = 18.78 / 26.40 \\ = 0.71$$

Critical Flaw Size

$$a_{Cr} = K_{IC}^2 \times Q / (1.21 \times P_i \times (\sigma_{\text{total}})^2) \\ = 136^2 \times 0.8 / (1.21 \times 3.14 \times 18.78^2) \\ = 11.04 \text{ mm or } 2a = 22.08 \text{ mm}$$

$$a/_{2C} = 0.05 \Rightarrow 2C = 11.04 / 0.05 \text{ i.e. } 2C = 220 \text{ mm}$$

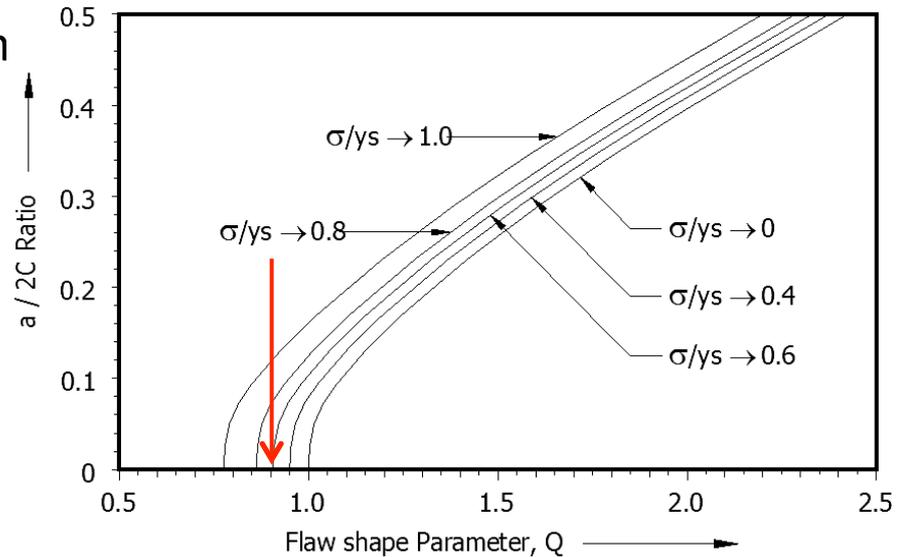


Fig. Flaw Shape Parameter Curves for Surface and Internal Cracks

$$\text{where, } Q = \phi^2 - 212 (\sigma / \sigma_{ys})^2$$

Summary

All anticipated damage mechanisms	
Visual abnormality	No significant visual abnormality
Structural distortion	No significant bulging No change in verticality
HTHA	Did not show significant damage.
High temp. corrosion	No reduction in thickness
Microstructural properties	No significant degradation is observed from external surface. Grain size ASTM 9 to 10 everywhere
Weld joints	No defect observed in WFMPI Defect at CS3 has dimensions less than critical size
Simulation study	Heat simulation indicated the overall strength as acceptable as per minimum requirement of SA 516 gr 70
FFS calculations	The flaw at CS3 is acceptable considering FFS calculations

Judgment of FFS

- From the accessible inspection and simulation studies it is concluded that the reactor has not been affected due to short term exposure to 710°C temperature to an extent that it is of immediate concern. The condition of reactor vessel is considered fit-for-service, for further operation as per OEM design and operation guidelines. Monitoring of flaw size at CS3 weld joint is to be done within next 2 years of operation.
- Considering the limitation of the inspection which excludes internal side of the reactor, regarding distributors, support trays or fittings, no judgment on their internal condition could be provided.

Leak before break Assessment of storage tank of Ammonia

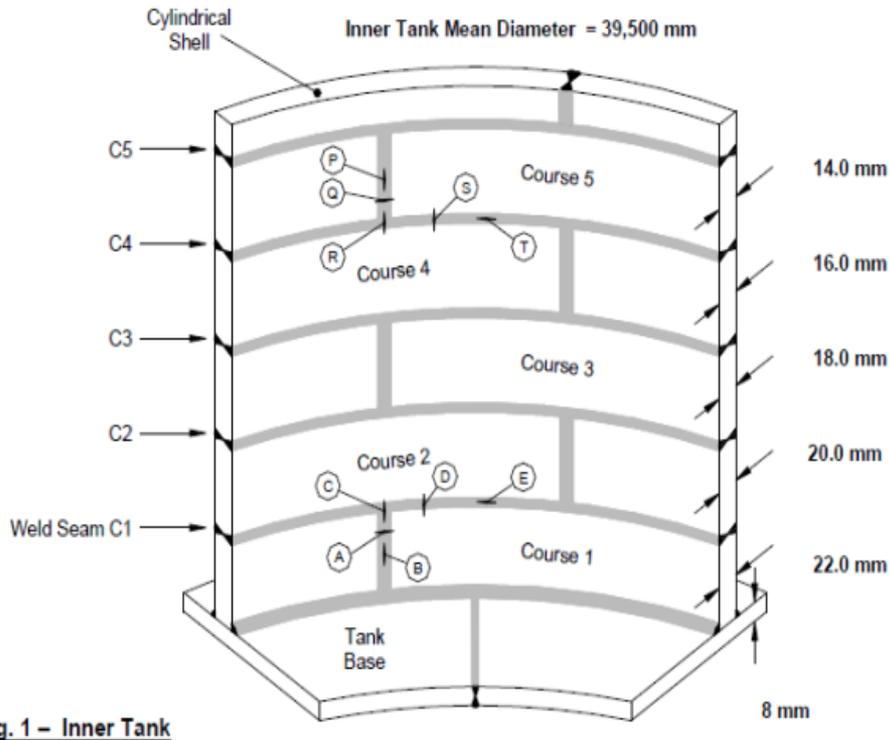


Fig. 1 – Inner Tank

- 18000 mt liquid NH₃
- Design Code:-API 620 Appendix-R, 1978
- Single wall integrity
- Refrigerated liquid ammonia atmospheric storage tank
- Inner Tank A-537 Class-1; Outer Tank IS-226
- Year of Construction:- 1983 (as per 1996 inspection report)
- 1st to 7th course vertical & circ seams 100% Radiography. The NDT for the remaining seams was as per code.

Inspection history

- In 1996, after 13 years in service, the first internal inspection of the inner tank weld seams and floor plate welds using MPI technique revealed no SCC type defects. Also the inner tank internal surface was free of corrosion, based on visual inspection and ultrasonic thickness survey.
- The Ultrasonic and MPI scope and coverage are detailed in the 1996 L inspection report

Approach for LBB study

Stresses due to applied loads

- Stresses in Inner Tank Cylindrical Shell under Product Condition
- Shell Stresses at Defect Tolerance Assessment Location
- Hydro test Stress & Residual stress at Weld Seams C1 and C4

Assessment Methodology

- Defect Assessment Locations & Defect Orientations
- Inner Tank Load Conditions (Liquid NH₃ Heights) consid

Material Properties

- Applied and Residual Stress Sensitivity Consideration of Input Data
- Summary of Input Data for Fracture Mechanics Calculations

Results of the Fracture Mechanics Assessments

Discussion of Results

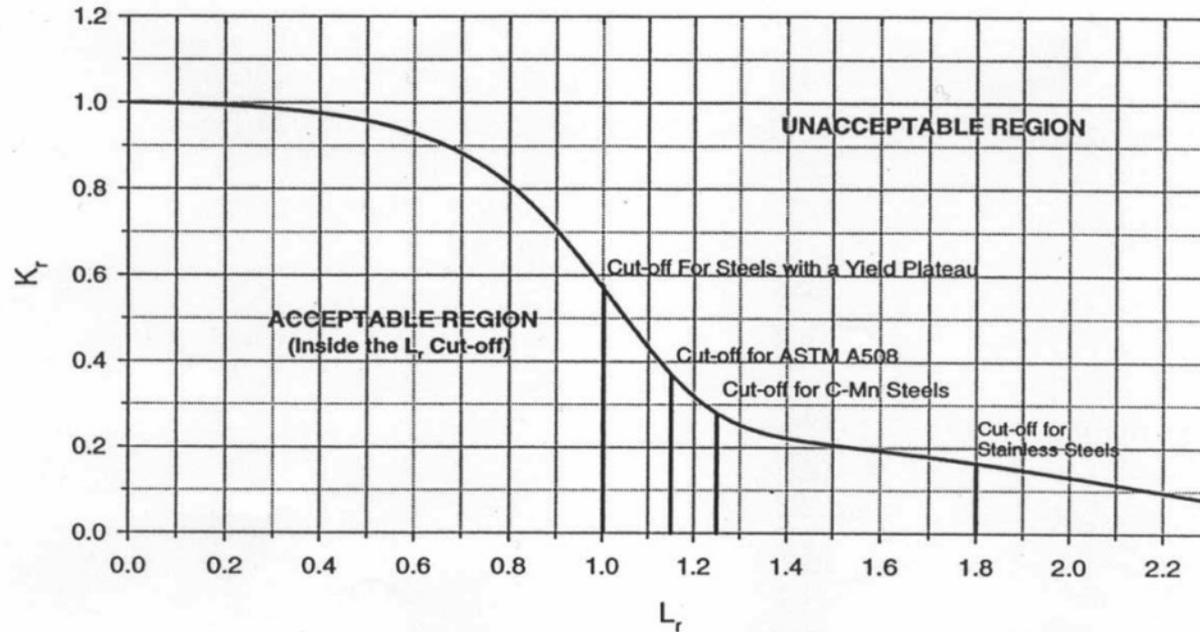
Stresses due to applied loads – inner tank

- In order to carryout the fracture mechanics assessment to determine defect tolerance, it is necessary to calculate the stresses due to applied loads (due to head of liquid and vapour pressure above liquid level) at the defect assessment locations. In this particular case, the relevant locations are the cylindrical shell courses 1 to 5.
- The applied loads considered are operating conditions and pre-commissioning hydro test conditions. The latter is necessary to estimate the relaxed welding residual stresses at the welds after a hydro test.

Consideration for SCC

- A point worth noting is that only a small percentage of the fully refrigerated anhydrous liquid ammonia storage tanks in the world have been known to have suffered from ammonia induced stress corrosion cracking [SCC]. In all such cases, only the lower parts of the tanks have been affected.
- Furthermore, in such cases, all SCC were found to be only located in the weld areas and were orientated transverse to the welds. (No SCC orientated parallel to the welds).
- Based on this experience, the assessment locations considered in this study have been carefully selected to evaluate the overall integrity of vulnerable lower parts of the tank, if such areas are susceptible to NH₃ induced stress corrosion cracking at -33°C .

Failure Assessment Diagram



Notes:

1. The FAD is defined using the following equation:

$$K_r = (1 - 0.14L_r^2) [0.3 + 0.7 \exp(-0.65L_r^6)] \quad \text{for } L_r \leq L_{r(\max)}$$



Questions?